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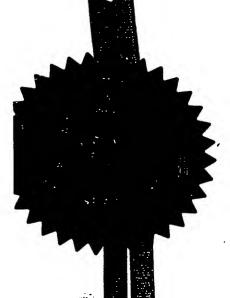
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APPARATUS AND METHOD FOR HEATING FLUIDS

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APPARATUS AND METHOD FOR HEATING FLUIDS

Certain improvements have been discovered which are here described and follow on the Apparatus and Method for Heating Fluids filed on July 4th, 2003 as Patent Application GB 0315576.9.

Figure 24 is a longitudinal sectional view of the heat generating device of Fig. 1. according to a fifth embodiment of the present invention, deploying a one-piece rotor and shaft component with axial feed port and internal intake passages.

Figure 25 is a longitudinal sectional view of the heat generating device of Fig. 24 with the modification in respect of the rotating unit of having a fluid throttle orifice disposed at the entrance to the fluid intake passageway.

Figure 26 is a longitudinal sectional view of the heat generating device of Fig. 24 with the modification in respect of the rotating unit of having one or more angled passageways in the interior of the rotating unit.

Figure 27 is a longitudinal sectional view of the heat generating device of Fig. 26 with the modification in respect of the rotating unit of having one or more fluid throttle orifices disposed in the interior of the rotating unit and communicating with one or more holes in the first row of holes

Figure 28 is a longitudinal sectional view of the heat generating device of Fig. 1. according to a sixth embodiment of the present invention, deploying a two-piece rotor and shaft component, where an axial feed port opening to a longitudinal passageway is arranged to be connected with more than one row of holes opening on the periphery of the rotor, via directly or via a fluid throttle orifice.

Detailed Description of the Fifth Illustrative Embodiment of the Invention

As this embodiment of the present invention, depicted in Fig. 24, has a rotating component 200 having an entrance port 201 leading to internal longitudinal passageway 202. Passageway 202 connects with one or more radial passageways 205 which direct the fluid, entering at intake 10iv to the exterior peripheral surface 206 that lies radially inwards of bore 207. Once fluid entering this annular clearance at the point where the radial passageways 205 open 210 on peripheral surface 206, the fluid travels across a series of rows of holes denoted by reference numerals 211, 212, 213 before exiting the device in a heated condition at threaded exit connection 214. The realtively cold fluid entering at axial port 201 picks up heat from the rotating component 200 during its transit to opening 210 on peripheral surface 206, thereby pre-heating the fluid.

As compared to Fig. 24, Fig. 25 incorporates a fluid throttle 218 at the inner end 219 of rotating unit 220, the throttle has a relatively small central hole which acts as an orifice to slow down the flow rate of fluid from inlet 10iv to the longitudinal passageway 222.

As compared to Fig. 24, Fig. 26 has at least one inclined passageway 230 connecting with longitudinal passageway 237 on the one hand, and opening at 231 in the volume space between wall 235 of housing 2 and face 236 of the rotating component 234. The relatively cold fluid entering the device at inlet 10iv near axial port 238 flows through the passageways 237, 230 before being redirected at opening 231 to flow radially outwardly in the volume space between wall 235 and face 236 to reach annular clearance where a number of holes 240 are positioned along the exterior surface of the rotating unit 234. The heated fluid exits the device at threaded exit connection 214.

As compared to Fig. 24, Fig. 27 shows a pair of fluid throttle 250, 251 disposed in the rotating unit 252. The rotating component 252 has an entrance port 253 leading to internal longitudinal passageway 254. Passageway 254 connects with radial passageways 255, 256 to communicate via respective throttles 250, 251 with holes 260, 261 of the first array of holes in the rotating unit.

Detailed Description of the Sixth Illustrative Embodiment of the Invention

In this embodiment of the present invention, depicted in Fig. 28, the rotating component comprises a rotor sleeve portion 270 fixed to a drive shaft portion 271, preferably by a heat-shrink fit.

The exterior of the rotor sleeve 270 is shown having a conical male exterior surface 272 and surrounding sleeve housing member 275 is provided with a complimentary opposing female conical surface 176.

However, it should be noted that either or both rotor sleeve 270 and sleeve housing member 275 may be cylindrical.

As shown, there are fours rows comprising a series of openings or depression zones, here in the formed of drilled holes such as holes denoted by reference numerals 280, 281, 282, 283. By way of example, all such holes 280-283 shown lying about rotational axis 290 are connected by respective smaller holes 290, 291, 292, 293 to longitudinal pasageway 295. By way of further example, the holes shown lying below the rotational axis are connected to longitudinal passageway via a respective throttle, shown for example as throttle 296.

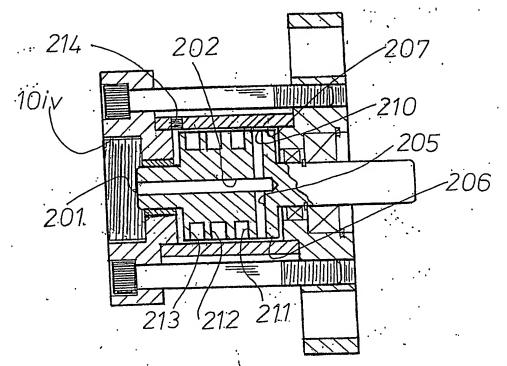
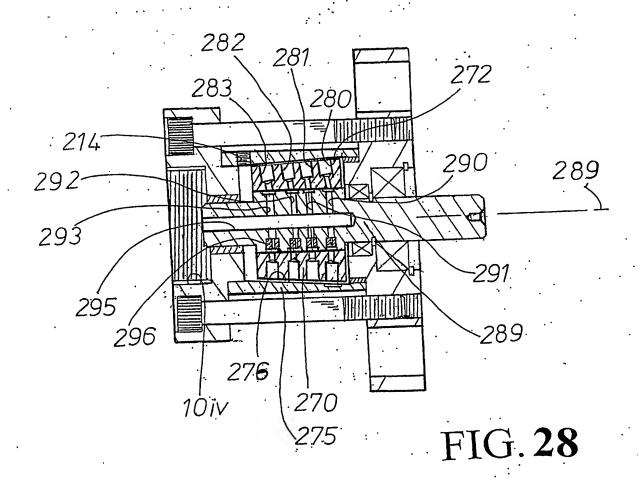


FIG. 24



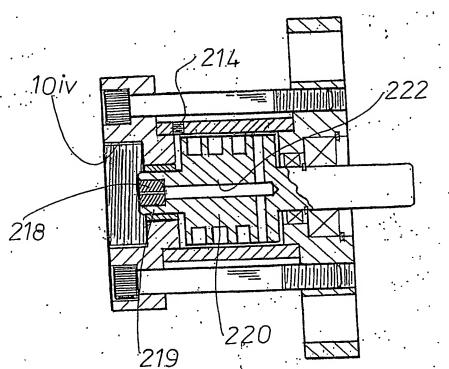
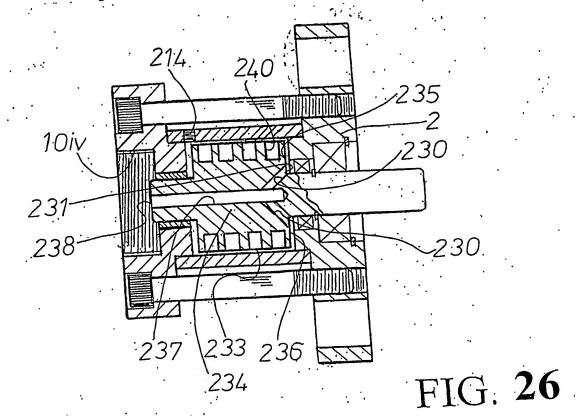


FIG. 25



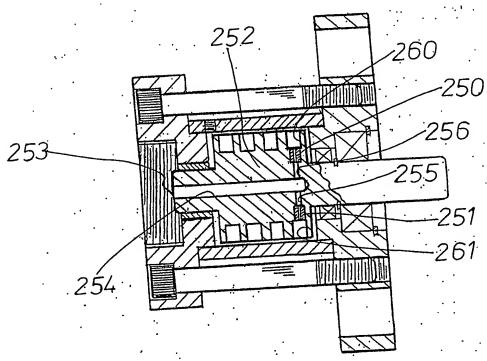


FIG. 27